



## THERMAFLO ENGINEERING CULTURE PROBLEM SOLVING

- Manufacturer Systems that Save Energy and Last 30 Years Plus
- Use Steam at the Highest Efficiency Possible in Our Designs
- Educate our Customers and Representatives on Steam and Heat Transfer
- Bring Experience to The Table
- > Hands on Face to Face
- Designed for Steam

ASME Certified U and UM NB Stamp Holders

ASME Certified
Welders and
Hartford Boiler 3<sup>rd</sup>
Party Quality
Program

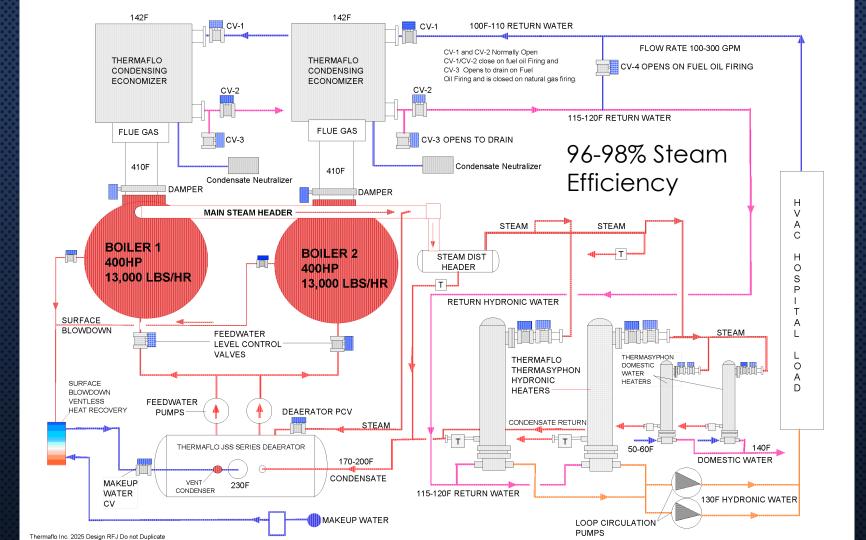
Innovative Long Lasting Designs

Innovation

Committed to R&D

**Training** 

THERMAFLO



Steam Basics
"A Blind Flash of the Obvious"

STEAM /
HOT WATER
AND HEATER
TRANSFER
SYSTEMS



FOLLOW ME

# Today We Are Going to Tell You What We Learned Working In the Field for 50 Plus Years



- ✓ Steam Has A lot More BTUs to Transfer When Used at Very Low Pressures
- ✓ Steam Traps Can be Your Best Friend or Worst Enemy.
- ✓ Most Steam Systems in the field are piped Wrong!!
- ✓ Condensate Pumps Fail because of Incorrectly Designed Steam Systems
- ✓ Steam Must Be Dry to Transfer Energy "I Thought all Steam was Dry"
- ✓ Wet Steam Kills Heat Transfer and Cost More to Use, Its Bad and You Have It, You're Engineering It, Designing It, and Using It!
- ✓ Any Flash Tank Vented to Atmosphere Waste Energy and Cost a Ton to Install What is a Flash Tank Anyway?
- ✓ Piping Systems are designed with Good Intentions but are not always installed that way in the field.
- ✓ Simple is Better, Less is Better, and It Starts with Design and Knowledge

## Anyone Seen This Before?

ITS SO COMMON AROUND THE COUNTRY THAT ITS CONSIDERED NORMAL!

ITS NOT NORMAL AND WITH PROPER ENGINEERING THIS CAN BE TOTALLY ELIMINATED

STEAM COST \$15.00 to PRODUCE PER 1000 Lbs! Up 25% Since Jan 1 2021

A Facility Generating 5,000 lbs/hr Avg Operating 300 Days A Year Pays \$540,000 Just in Fuel



These Are Familiar Sights in the Back of Hospital, University, College, and Industrial Maintenance Shops and Facilities

- ✓ Failed Control Valves and Isolation Valves
- ✓ Failed or Troublesome PRV Stations
- ✓ Failed Condensate Pumps / Seals and Motors
- ✓ Failed Heat Exchangers
- ✓ Failed Steam Traps
- ✓ Leaking Valves
- ✓ Wet and Damaged Insulation

THIS IS NOT NORMAL!



# SIMPLE STEAM FACTS YOU SHOULD KNOW

- Steam is created by adding \*\*Exist Energy to water until the water changes state and turns into a VAPOR.
- HEAT ENERGY IS EXPRESSED IN BTU'S. MORE BTUS MORE HEAT TO TRANSFER!
- 1 BTU IS THE AMOUNT OF HEAT ENERGY REQUIRED TO RAISE THE TEMPERATURE OF 1 POUND OF WATER BY 1°F.
- THE BTU CONTENT OF THE STEAM VAPOR IS CALLED LATENT HEAT LATENT HEAT IS THE WORKHORSE AND THIS HEAT YOU CANT
  READ ON A THERMOMETER BUT IT CHANGES WATER TO A VAPOR WE CALL STEAM!
- THE BTU CONTENT OF THE WATER (FEEDWATER AND CONDENSATE) IS CALLED SENSIBLE HEAT THIS IS THE HEAT YOU CAN READ ON A THERMOMETER!
- A LB of Water Contains 180 BTUS 32F to 212F NOW A LB of STEAM at 0 PSIG Contains 970 BTUs 5 TIMES THE BTU
  Energy!
- When Water is In Your Steam the Latent Heat Is Less! That Same LB of STEAM with 10% Liquid Entrained in the Vapor has
   873 BTUS of Latent Heat Per LB

## BASIC STEAM PROPERTIES CHART

0 psig Steam 212F Latent Heat Btus 970 Sensible Heat Btus 180 Volume per Ib 26.8 Cubic Ft per Ib

Look at Vacuum Steam at 10" 982 BTUS Per Lb Latent

100 psig Steam 338F Latent Heat Btus 880 Sensible Heat Btus 309 Volume per lb 3.89 Cubic Ft per lb

STEAM AT LOWER PRESSURES CONTAINS MORE BTUS THAN HIGH PRESSURE STEAM

					Specific						Specific
Gauge Pressure	Temper- ature		atin Btu/l		Volume Cu. ft.	Gauge Pressure	Temper- ature		at in Btu/l		Volume Cu. ft.
PSIG	°F	Sensible	Latent	Total	per lb.	PSIG	°F	Sensible	Latent	Total	per lb.
25	134	102	1017	1119	142.0	185	382	355	843	1198	2.29
OF 20 15	162	129	1001	1130	73.9	190	384	358	841	1199	2.24
	179	147	990	1137	51.3	195	386	360	839	1199	2.19
≧ 10 5	192 203	160 171	982 976	1142 1147	39.4 31.8	<b>200</b> 205	<b>388</b> 390	<b>362</b> 364	<b>837</b> 836	<b>1199</b> 1200	<b>2.14</b> 2.09
<sub>5</sub>	203 212	180	970	1150	26.8	210	390	366	834	1200	2.09
1	215	183	968	1151	25.2	215	394	368	832	1200	2.00
2	219	187	966	1153	23.5	220	396	370	830	1200	1.96
3	222	190	964	1154	22.3	225	397	372	828	1200	1.92
4	224	192	962	1154	21.4	230	399	374	827	1201	1.89
5	227	195	960	1155	20.1	235	401	376	825	1201	1.85
6	230 232	198 200	959 957	1157	19.4 18.7	240 245	403 404	378	823	1201	1.81
7 8	232	200	957 956	1157 1157	18.7	245 250	404 406	380 <b>382</b>	822 <b>820</b>	1202 <b>1202</b>	1.78 <b>1.75</b>
9	237	205	954	1159	17.1	255	408	383	819	1202	1.73
10	239	207	953	1160	16.5	260	409	385	817	1202	1.69
12	244	212	949	1161	15.3	265	411	387	815	1202	1.66
14	248	216	947	1163	14.3	270	413	389	814	1203	1.63
16	252	220	944	1164	13.4	275	414	391	812	1203	1.60
18	256	224	941	1165	12.6	280	416	392	811	1203	1.57
20	259	227	939	1166	11.9	285	417	394	809	1203	1.55
22 24	262 265	230 233	937 934	1167 1167	11.3 10.8	290 295	418 420	395 397	808 806	1203 1203	1.53 1.49
26	268	236	933	1169	10.8	300	420 421	398	805	1203	1.49
28	271	239	930	1169	9.85	305	423	400	803	1203	1.45
30	274	243	929	1172	9.46	310	425	402	802	1204	1.43
32	277	246	927	1173	9.10	315	426	404	800	1204	1.41
34	279	248	925	1173	8.75	320	427	405	799	1204	1.38
36	282	251	923	1174	8.42	325	429	407	797	1204	1.36
38	284	253	922	1175	8.08	330	430	408	796	1204	1.34
<b>40</b> 42	<b>286</b> 289	<b>256</b> 258	<b>920</b> 918	<b>1176</b> 1176	<b>7.82</b> 7.57	335 340	432 433	410 411	794 793	1204 1204	1.33 1.31
44	209	260	917	1177	7.31	345	434	413	793 791	1204	1.29
46	293	262	915	1177	7.14	350	435	414	790	1204	1.28
48	295	264	914	1178	6.94	355	437	416	789	1205	1.26
50	298	267	912	1179	6.68	360	438	417	788	1205	1.24
55	300	271	909	1180	6.27	365	440	419	786	1205	1.22
60	307	277	906	1183	5.84	370	441	420	785	1205	1.20
65	312	282	901	1183	5.49	375	442	421	784	1205	1.19
70 <b>75</b>	316 <b>320</b>	286 <b>290</b>	898 <b>895</b>	1184 <b>1185</b>	5.18 <b>4.91</b>	380 385	443 445	422 424	783 781	1205 1205	1.18 1.16
80	324	294	891	1185	4.67	390	446	425	780	1205	1.14
85	328	298	889	1187	4.44	395	447	427	778	1205	1.13
90	331	302	886	1188	4.24	400	448	428	777	1205	1.12
95	335	305	883	1188	4.05	450	460	439	766	1205	1.00
100	338	309	880	1189	3.89	500	470	453	751	1204	.89
105	341	312	878	1190	3.74	550	479	464	740	1204	.82
110 115	344 347	316 319	875 873	1191 1192	3.59 3.46	600 <b>650</b>	489 <b>497</b>	473 <b>483</b>	730 <b>719</b>	1203 <b>1202</b>	.75 <b>.69</b>
120	350	319	873 871	1192	3.46	700	505	<b>463</b> 491	719	1202	.64
125	353	325	868	1193	3.23	750	513	504	696	1200	.60
130	356	328	866	1194	3.12	800	520	512	686	1198	.56
135	358	330	864	1194	3.02	900	534	529	666	1195	.49
140	361	333	861	1194	2.92	1000	546	544	647	1191	.44
145	363	336	859	1195	2.84	1250	574	580	600	1180	.34
150	366	339	857	1196	2.74	1500	597	610	557	1167	.23
155	368	341	855	1196	2.68	1750	618	642	509	1151	.22
160 165	371 373	344 346	853 851	1197 1197	2.60 2.54	2000 <b>2250</b>	636 <b>654</b>	672 <b>701</b>	462 <b>413</b>	1134 <b>1114</b>	.19 <b>.16</b>
170	375	348	849	1197	2.47	2500	669	733	358	1091	.13
170	070	0-10	040	1107	2.71	2000	000	700	000	1001	.10

## **Understanding Steam Demand**

 $Q = L \times Delta T \times C \times 500 \times Sg = BTUS Required$ 

H = Latent Heat of Steam

Q = Steam Load lbs/hr

L = GPM

DT = Temp Rise

C = Specific Heat

 $500 = 60 \text{min/hr} \times 8.33 \text{ lbs gallon}$ 

Sg = Specific Gravity

H = Latent Heat of the Steam

Used

Short Cut Version: GPM divided by 2 x Delta T x 1.10 = lbs/hr Steam Example: 400 GPM Water Heated from 140F to 180F using 15 psig Steam

 $400 \times 40F \times 1 \times 500 \times 1 = 8,000,000$  BTUS

945 BTUS per lb 15 psig

 $10,000 \times .002 = 20 \text{ GPM}$ 

 $= 8,465 \, lbs/hr$ 

Same Application using 1 psig steam with 968 BTUs per lb = 8,264 lbs/hr A Difference of 201 lbs/hr for the Same Work!

Realize that 15 psig Steam also generates 4% wasted Flash Steam or 330.56 lbs/hr!! Thru the Vent

1 Boiler HP = 34.5 Lbs/hr of Steam 100 Boiler HP= 3400lbs/hr of Steam x Eff Rate Generally .85 So 2,890 lbs/hr with Eff Rate Applied Lbs/hr of Condensate x .002 = GPM Flow



## WHAT IS A STEAM TRAP?

STEAM TRAP IS A DOOR!

WHEN CONDENSATE COMES TO THE DOOR THE DOOR OPENS

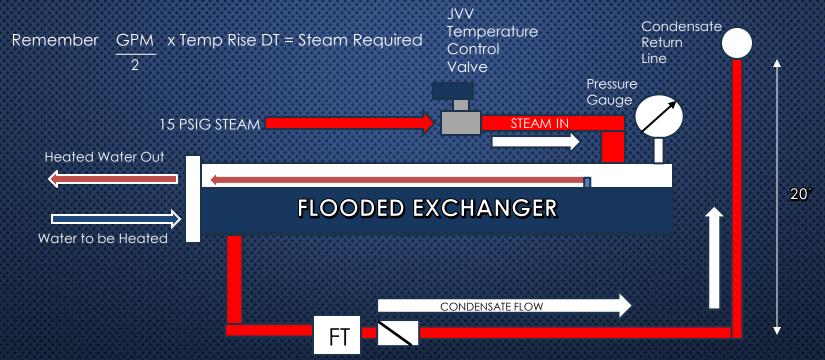
THIS ALLOWS THE LOW
BTU CONDENSATE TO
LEAVE THE STEAM
SPACE AND LATENT
HEAT TO DO THE
WORK!





WHEN STEAM COMES TO THE DOOR THE DOOR THE DOOR CLOSES HOLDING BACK THE STEAM AND ALLOWING THE LATENT HEAT TO HAVE TIME TO TRANSFER LATENT HEAT BTUSS AND DOING THE HEAT TRANSFER WORK

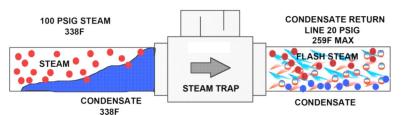
ALL STEAM TRAPS MUST HAVE MORE INLET PRESSURE THAN OUTLET PRESSURE TO OPERATE!



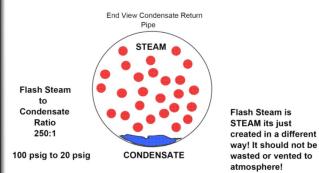
When the JVV Modulates with flow demand to hold temperature the pressure in the shell starts to drop as the steam demand decreases. When the pressure in the shell reaches 8 psig and lower the steam trap stops working and condensate backs up in the shell. This causes waterhammer on the tubes and thermal shock.

It takes 1 psig pressure to push or lift condensate 2.3' vertical. So to push the liquid condensate up the 20' you need 8.69 psig.

Flash Steam is created when Condensate under High Pressure and a High Temperature is discharged to a return line that is at a lower pressure and temperature as illustrated below.



The higher the inlet pressure and the lower the outlet pressure the higher the % of Flash Steam Produced



### FLASH STEAM TABLE

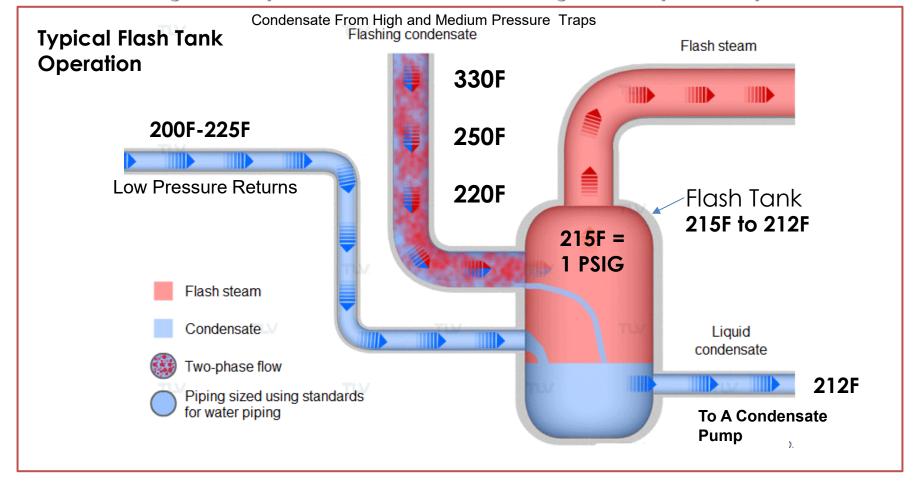
Pressure	% Flash Steam per lb			
PSIG	0 psig	20 psig		
5 15 30 100 125	1.7 4.0 6.5 13.3 14.8	0 0 1.7 8.8 10.3		

Example: 1000 lbs/hr of 100 psig discharging to 0 psig (Atmosphere) will produce 13.3% Flash or 133 lbs/hr of Flash Steam

# THE FLASH STEAM STORY

Flash Steam occurs when a steam trap opens and discharges condensate from a steam line into a return line operating at a lower pressure. This occurs at all pressures even at 15 psig and lower. Higher the steam pressure the more flash steam is produced STEAM IS STEAM

Flash Tanks are Designed to Separate Condensate from Flashing Steam Vapor and Equalize all Pressures



```
Condensate Recovery Savings Analysis
A. Condensate (Lost to Drain) A= 5,000 lbs/hr
   Annual Hrs of Operation a Year B= 7200
C. Total Water Cost (C1 + C2)
   C1 Untreated Water and Sewer .005 $per Gallon
   C2 Water Treatment Chemicals .010 $per Gallon
                               C = .015 $per Gallon
D. Makeup Water Preheating Cost of Feedwater
D1 Condensate Return Temperature: 190F
D2 Makeup Water Temperature: 60F
D= (D1-D2) \times 1.0BTU/ LB/F D= 130 BTUs/Lb
E = Steam Cost
                               E= $15.00 $1000/lb
F Savings in Water Cost
(A) 5000 \times (B) 7200 \times (C) .015
                               = $64,748.00
       8.34 LB per Gallon
G Savings for Heating Makeup Water
(A) 5000 \times (B) 7200 \times (D) 130 \times (E) $15.00 = $70,200.00
               1000 x 1000
Total Annual Savings F + G
```

TOTAL ANNUAL SAVINGS FOR RETURN \$134,948.00

F \$64,748.00 + G \$70,200.00

Proper Return of Condensate
Is Important
This Payback Analysis
Shows that 4 GPM
Going to Drain and Not
Being Returned Cost
\$134,948.00 Yearly

Of Steam Output Should
Be as High as Possible!

### STEAM PROPERTIES & FLOW CHARACTERISTICS

### SIZING STEAM PIPES . Steam Velocity Chart (Schedule 40 pipe)

Saturated steam lines should be sized for a steam velocity of 4800 to 10000 ft/min.

Piping on pressure reducing stations should be sized for the same steam velocity on both sides of the regulator. This usually results in having a regulator smaller than the piping and having larger piping on the downstream side of the regulator.

### **Example using Steam Velocity Chart:**

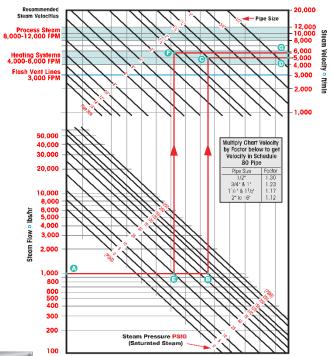
100 PSIG Inlet Pressure to control valve; 25 PSIG Outlet Pressure; 1,000 lbs/hr flow rate; Determine pipe size required.

### Upstream Pipina:

Enter Velocity Chart at (\$1000 lbs/hr. Follow line to (\$100 PSIG Inlet Pressure. Follow line vertically upwards to (\$11/2" Pipe Diameter. Steam Velocity at (\$\text{\$\text{\$0}\$}\) shows 4800 ft/min.

### Downstream Piping:

Enter Velocity Chart at (2) 1000 lbs/hr. Follow line to (3) 25 PSIG Outlet Pressure. Follow line vertically upwards to (3) 21/2" Pipe Diameter. Steam Velocity at (6) shows 5500 ft/min.



(Saturated Steam)

Note: Cendensote Line & Vent Line Size based on using Schedule 40 Pipe

Thermafio Engineering Inc. Newberry, S.C. 704-940-1228

# Is Steam Line and Water Line Sizing Important?

# YOU BET!

Under Sized Steam and Water Lines Cause:

- Control Valves to be Undersized
  - Poor Heat Transfer
  - High Pressure Drops
  - Poor Equipment Performance



# REMOVING THE CONDENSATE FROM THE STEAM SYSTEM BY DESIGN

- > Condensate Cuts Valve Seats
- Condensate Erodes Steam Piping
- > Condensate Kills Steam Heat Transfer
- > Condensate in a Steam Line causes Water Hammer and Thermal Shock
- Poor Steam Quality Drives the Cost of Steam

# What is the Quality Of Your Customers Steam?

Is Your Steam Dirty? Is Your Steam System Old? How Far are Your Users from the Boiler? Are Your Main Drips Located every 150 to 300'? **Are Your Steam Traps Working?** Do Your Have Steam Separators in Your Lines? Are Your Steam Main Drips Built Correctly?

# Steam Quality



Is Steam Quality a Big Deal? Most people never talk about this or consider this!

- Most all Valve Failures are caused by Steam Quality and Incorrect Piping Install.
- ✓ Wet Steam or Condensate In Steam flowing across valve seats at High Velocities wearing away metal as if flows. This causes valves to leak and blow by steam where shutoff is required.
- Condensate Buildup causes flashing at the valve outlet when pressure drop occurs damaging seats and packing in valves.
- Wet Steam Reduces the Heating Value of a Lb of Steam
  Steam Exits the Boiler at 98-99% Dry BUT as it flows into the piping
  system and out to the users 200-3000' away condensate forms in
  the steam and reduces its Dryness Fraction or ability to transfer latent heat
  vapor energy. Average Steam Quality is 85-90% as it reaches users. This means
  85 to 90% of the Latent Heat Value can be transferred so MORE steam in lbs/hr
  is required for the duty. Simply put its cost more to heat your process!

As the Steam Flows through Main Supply Piping it Latent Heat transfers to the pipe walls and through the Insulation to Atmosphere. When this happens small water droplets form and are carried with the flowing STEAM

## STEAM FLOWING ON TOP OF THE CONDENSATE

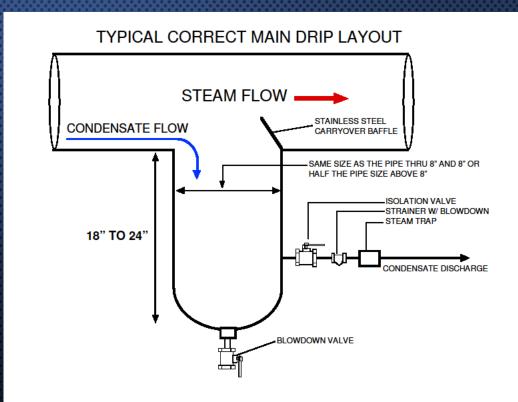
CONDENSATE LIQUID IS HEAVIER THAN STEAM VAPOR AND FLOWS ON THE BOTTOM OF THE STEAM MAIN PULLED BY THE STEAM FLOW

If the condensate is not properly removed from the Steam Vapor line it reduces the efficiency of the steam because (Condensate) water have far less BTU value per lb Than the STEAM 7-10 Times Less! In addition condensate buildup cause piping

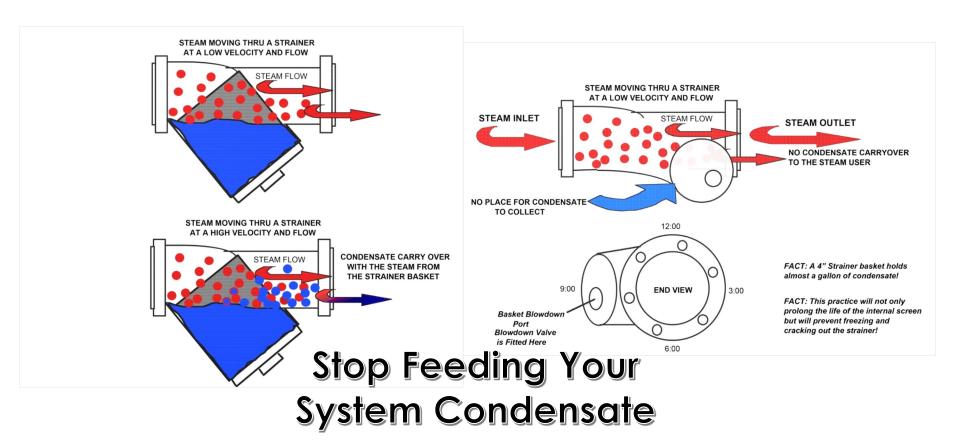
## QUALITY STEAM STARTS IN THE SUPPLY

INSTALL MAIN DRIPS EVERY 150' MIN
END OF MAINS
CONTROL VALVE INLETS
OR PIPING CHANGE IN DIRECTION

- POOR QUALITY STEAM DECREASES HEAT TRANSFER NO BTUS IN CONDENSATE!
- POOR QUALITY CUTS THE SEATS ON CONTROL VALVES
- POOR QUALITY STEAM WILL DEPOSIT SCALE
   ON THE TUBE SURFACES AND SEATS ON
   CONTROL VALVES



### CORRECT STEAM STRAINER INSTALLATION IN FRONT OF CONTROL VALVES



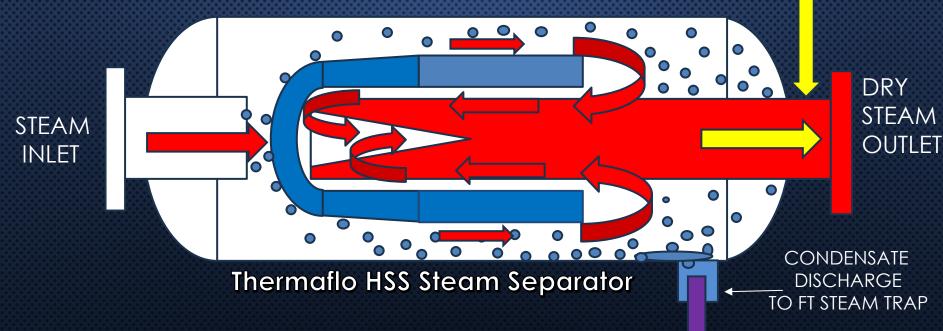
Example: Hydronic Heating Converter 200 GPM 140F to 180F REQUIRES 2,000 LBS/HR 15 PSIG Steam 2,000,000 BTUS/Hr to the Control Valve inlet and 5 psig in the Shell

5 psig steam @ 227°F @ 100% Dry (saturated) steam supplies 961 Btu / lb. requires **2,081.16 lbS/hr**.

@ 90% Dryness Fraction, steam supplies 864 Btu/lb now requires 2314 /hr. A 11% Increase



The discharge steam flow is dry and with high BTU quality with 98% of the condensate removed





Globe Control Valve

### Globe Control Valves Vs V Ball Style

- Small CV Flow Factors 2" CV 42 versus a 167 CV of a V Ball
- Flow Turndown is 30: 1 versus 300: 1 plus on a V Ball
- Tight Shutoff is Only Achieved when a Soft Seat Option is Selected Dead Tight Shutoff is Standard on a V Ball Style
- A standard 3" Globe Valve weighs 91 lbs and has a CV of 73 By comparison a 2" V Ball weighs in at 22 lbs and has a CV of 167
- Electronics are not slow anymore!
   8 Second Speed actuator less than 1 second between 10% movement
- V Balls have been used in Industry since the 1970s

V Ball Control Valve





# Why Condensate Return Pumps are a Major Issue







Myth 1

It's a Common Practice and Belief that Steam Traps can be piped from any Pressure directly into a Condensate Pump Receiver!

# Myth 2

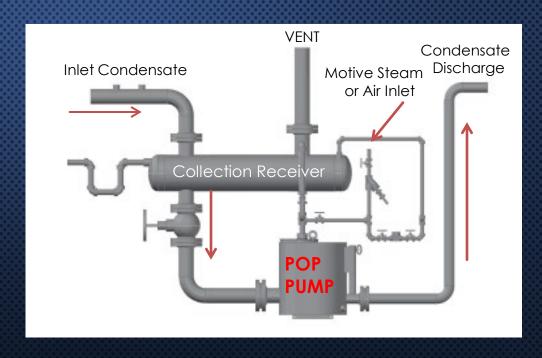
Its also a Common Belief that you can install a Flash
Tank Pipe all Condensate to the Flash Tank and the Outlet Temperature
Will Automatically Drop to 180F and Cavitation on the
Condensate Pumps will be Eliminated!
Flash Tank Vents are Expensive and Discharge Wasted BTUs

## THE ANSWER TO ELECTRIC PUMPS!



Pumps Condensate to 400F
ASME Stainless Steel Body
No Electric Motors or Panels
No Seals or Packing
No NPSH Issues

# POP Pressure Powered Condensate Pumps Non Electric Condensate Return



Simple Design 5 Million Cycle Guarantee of 5 Years from Installation

Capacities to 100,000 lbs/hr



# Steam Mechanical Room 2025 What if You Could:

Eliminate the Steam PRV Stations Eliminate the Flash Tanks 75% Fewer Steam Traps 75% Less Piping Eliminate Flash Tanks and Vents Eliminate Condensate Pumps Eliminate 80% of all Radiant Heat Loss Reduce Steam Usage 20%



## Zero Flash Heat Exchangers Save Energy for the End User

- Steam Quality in a Zero Flash is 100% as Compared to 85-90% In Any Conventional Steam Heat Exchanger System Due to the Inlet Steam Conditioning Design
- Steam is Used at a Very Low Pressure Which is Safe and Has A Latent Heat/Sensible Heat Content per LB of Over 1000 BTUs Per Lb of Steam Versus 865 for a Common Exchanger System
- Zero Flash Exchangers Utilize the Sensible Heat of the Steam Reducing The Overall Amount of BTUS needed for the Same BTU Duty EFFICIENCY OF 25% ANY STANDARD STEAM EXCHANGERS

# THERMA FLO

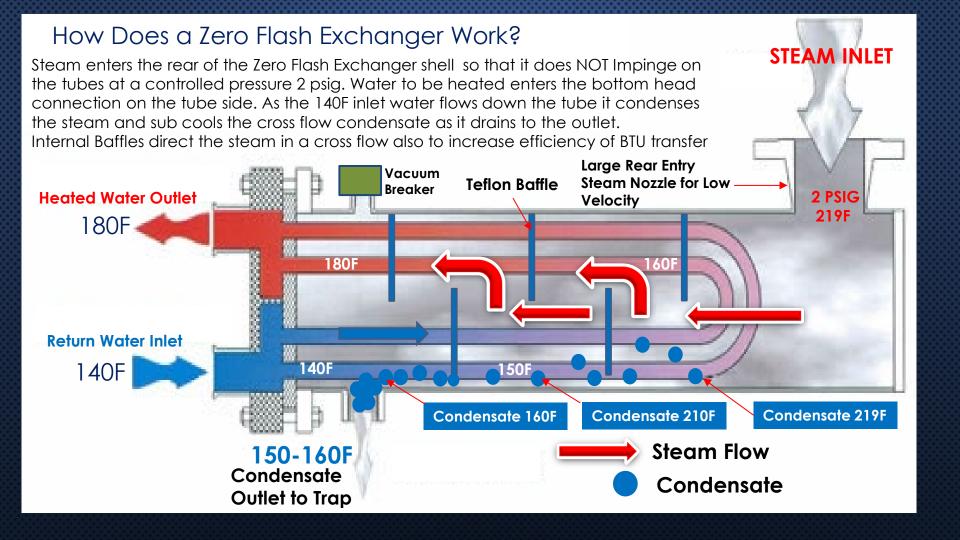
## Zero Flash Steam Fluid Heater System Open Loop

- Eliminates Wasted Flash Steam to Atmosphere and Flash Tanks
- Accepts High Pressure or Low Pressure Steam Subcools Condensate for High Efficiency
- System is Fully Pre Piped with all the Features Such as: Inlet Isolation Valve, Strainer

(installed correctly) Steam
Conditioning Station to remove all condensate and enhance Steam
Quality to 100%, JVV V Ball 300:1
Control Valve, FT Steam trap and the Zero Flash ASME Condensing
Heat Exchanger



Save 50% on Install Savings By Eliminating PRV Stations, Safety Valve Vents, Flash Tanks, Special Design Condensate Pumps



# Ask Yourself:

# TH500PH ThermaSyphon "Pumping Exchanger"

What if we could eliminate Steam PRV Stations from my mechanical room and campus or greatly reduce the size of them?

What if we could reduce the overall steam required to heat water over 15% across the board?

What if we could eliminate condensate return pumps?

What if we could eliminate open loop flash tanks, holes cut through roofs,

costly vent piping, and wasted flash steam to atmosphere?

What if we could reduce equipment space in a mechanical room 50%?

What if our exchanger equipment was designed for 30 Year service?

# THERMA FLO

# The Answer is Yes! ThermaSyphon

- The ThermaSyphon is a time proven shell and tube vertical space saving heat exchanger constructed of 316L High Alloy Stainless Steel to ASME Section VIII 150 psig
- ➤ High or Medium Pressure Steam Supply is reduced and controlled to low pressure in the shell at the Inlet and features a inlet Vertical Down Steam Seperator
- The ThermaSyphon features an inlet "Steam Vaporization Nozzle" to enhance the Shell "Steam Quality" to above a 98% dryness fraction saving 15 to 20% over any standard heat exchanger design
- The controlled constant shell steam pressure is used as a motive driving force to pump and return condensate to a central return line eliminating system Stall and those external condensate return pumps, and flash tanks
- The ThermaSyphon is equipped standard with a EC1000 UL Listed Touchscreen Controller that features slow ramp system startup of steam, controls shell pressure and outlet water temperature plus handles all safety alarms and controls!





## APPOX.  ## APPOX. #
## 18 AND STATE OF THE PROPERTY OF THE PROPERT
120   120
DO NO SUBJECT EQUIPMENT DESCRIPTION TO CONTROL TO CONTR
DO NO SUBJECT EQUIPMENT DESCRIPTION TO CONTROL TO CONTR
Thereacylymon Pumping Exchanger Features and Beautiful  - Design Controller Pressure
TEMPERATURES OR QUITDOR CONDITIONS  TEMPERATURES OR QUITDOR CONDIT
Political Constant Processors "Prompting Exchanges" places   Security   Secur
Polity (A)  Polity
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Manufacturers Of Steam Fired Specialty Equipment
*** "HH"   2880 Fair Avenue, Newberry, SC 29108   (704) 940-1228   www.thermafloengineering.com
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LEFT SIDE VIEW
Thermasyphon is a Patent Pending Design and Copyright of Thermaflo Inc FRONT VIEW RIGHT SIDE VIEW DIM TOL= +/- 1/2* CUSTOMER: J08# QIY. 10-30-25 The
THIS DRAWING AND INFORMATION THEREON ARE THE PROPERTY OF THERMAN-LO INCORPORATED AND SHALL NOT BE USED OR DISCLOSED, EXCEPT IN ACCORDANCE WITH PRIOR WRITTEN APPROVAL THIS DRAWING AND BILL OF MATERIAL REPRESENTS THE ENTIRE CONTENT OF SCALE: DRAWIN; PO# PLAN# FILE: REV.# PAGE:
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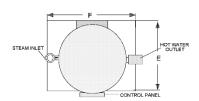
# Thermaflo TH750TM Steam Fired Water Heater

- Designed for Domestic Water Use Mainly Steam in the Tubes / Water in the 316L ASME Code Section VIII Shell
- Vertical Short Shell Spacing Design Less than 72" OAH
- Temperature is Controller to 140F or below with a Self Contained Thermostatic Mixing Valve ASSE 1017 Certified
- Inlet Steam BVV Safety Shutdown Valve
- EC100 UL Listed Safety Steam Shutdown System on High Temperature with BMS Dry Contact Alarm Notification
- Constant 15 psig Steam Pressure on the tubes eliminates condensate back, water hammer and thermal shock
- > Double Wall or Single Wall Inner Tube Bundle Construction
- High Capacity Float and Thermostatic Steam Trap
- Shell Pressure Gauge and Thermometer
- Vacuum Breaker and Inlet Steam Pressure Gauge
- Vertical Base Stand and Insulation Jacket





### THERMOMETER THERMAFLO 1' RECIRCULATION Made in USA CHECK VALVE 1" SHELL SIDE BLOWDOWN VALVE (4) VACUUM BREAKER 2" SHUTDOWN VALVE PRESSURE GAUGE 30"-0-30 PSIG FLOAT THERMOSTATIC US GAUGE 1981 STEAM TRAP 1.5 FTH46815 FRONT ELEVATION VIEW



#### CONDITIONS OF SERVICE DIMENSIONS INLET STEAM PRESSURE: 15 psig INLET TEMPERATURE: 40F OUTLET TEMPERATURE:140F С В D FLOW RATE 60 75 40 18 23 24 24 STEAM FLOW RATE:3000 LBS/HR

NOTES: 1. UNIT HYDROTESTED BEFORE SHIPMENT.

TOP VIEW

- 2. UNIT SHOWN IS VERTICAL STAND MOUNTED
- 3. 115 VOLT/1PHASE/60HZ POWER REQUIRED FOR OPERATION. 4. MOUNTING FRAME PRIMED AND PAINTED
- 5. CONTROL PANEL NEMA 4 UL LISTED.

Manufacturing Facilities: 2880 Fair Avenue Newberry, SC 29108 www.thermaflofloengineering.com Phone: 704-940-1228

### THERMAFLO TH750TM STEAM FIRED WATER HEATER STANDARD FEATURES AND OPERATION

The Thermaflo TH750TM Domestic Water Heater is a completely pre-piped and tested steam water heating system. Inlet supply steam is piped to the inner tube bundle at a constant pressure heating the incoming domestic water. Outlet domestic water demand automatically modulates the "Thermostatic Self Contained Thermal Valve" blending precise amounts of hot and cold water instantly to maintain is set point for all flow demands.

Outlet heated water temperature settings (110F to 150F) are easily made with a dial adjustment on top of the thermal control valve. Domestic loop water is recirculated to the recirculation port to maintain constant loop supply temperature The inlet steam supply valve will shutdown in case of overheating condition with dry contact BMS system output, and red alarm light on the front of the UL Listed control panel. Bacnet output available as an option.

Constant pressure steam is used as a motive condensate removal pumping force, so the TH750TM can be used as a pumping exchanger for condensate return, eliminating a condensate return pump in some applications

- ASME CODE SECTION VIII DIV1 U STAMPED CONST
- THERMOSTATIC SELF CONTAINED DOMESTIC ASSE
- 1017 WATER TEMPERATURE CONTROL VALVE ■ VERTICAL CROSS FLOW DESIGN EXCHANGER
- 316L STAINLESS STEEL SHELL CONSTRUCTION
- HEAVY WALL COPPER TUBE BUNDLE WITH TEFLON SEGMENTED BAFFLES
- SHELL BLOWDOWN VALVE AND EXPANDABLE TUBE DESIGN TO REMOVE INTERNAL SCALE BUILDUP
- ACCURACY +/- 4F UNDER NORMAL LOADS
- STEAM SAFETY HIGH LIMIT SYSTEM
- UL LISTED CONTROL 120V/1/60 PANEL
- VACUUM BREAKER AND PRESSURE GAUGE OUTLET WATER BIMETAL THERMOMETER
- HIGH CAPACITY FLOAT AND THERMOSTATIC STEAM TRAP
- COMPLETE PIPED AND TESTED PACKAGED SYSTEM
- 1 5" THICK INSULATION SHELL REMOVABLE JACKET

HEAT EXCHANGER DESIGN STANDARDS						
	MAXIMUM WORKING PRESSURE PSIG	MAXIMUM TEMPERATURE	TEST PRESSURE			
SHELL	150 PSIG	375F	250 PSIG			
TUBE SIDE	150 PSIG	375F	250 PSIG			
ASME CODE SECTION VIII, DIV 1 STAMP U/UM						

### CONNECTIONS

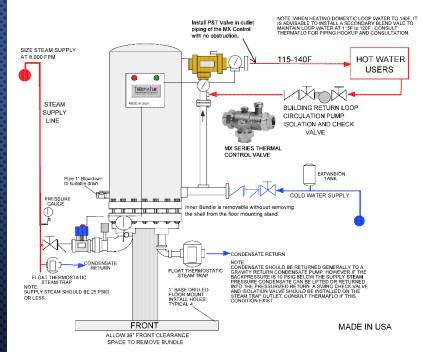
- 1 2" STEAM INLET SUPPLY NPT 15 PSI
- 2 1.5" CONDENSATE OUTLET 3 2" 140F HEATED WATER OUTLET 150LB FLG
- 4 2" COLD WATER INLET 150LB FLG

### MATERIALS OF CONSTRUCTION

SHELL	316L STAINLESS STEEL
TUBESHEET	304L STAINLESS STEEL
TUBES	COPPER DOUBLE WALL
BAFFLES	TEFLON 25% SEGMENTED
STEAM HEAD	CLOSE GRAIN CAST IRON



### TYPICAL PIPING AND INSTALL ARRANGEMENT



### PIPING NOTES AND MAINTENANCE FEATURES

- The water side domestic piping standard is 316L stainless steel for long service life.
- 2. The Thermaflo TH750TM Shell carries a 20 Year Warranty as a standard.
- 3. Each Thermaflo TH750TM is factory tested on live steam and water flow conditions before shipment, with outlet water setpoint verified.
- 4. Piping/Valves/and Exchanger meet and exceed NSF61 material requirements for domestic water service.
- 5. The internal heating tube bundles are fully removable for maintenance or repairs without dismantaling domestic water piping or removing the shell from the base stand.
- 6. The Thermal Control Valve can be removed by means of union bronze connections, and cleaned quickly when required or replaced after years of constant service.
- 7. Inner tube bundle teflon baffles are arranged so that optimum heat transfer is realized and expansion and alignment assured so that wear holes are not a failure point

Thermaflo Engineering Inc. Manufacturing Facilities: 2880 Fair Avenue Newberry. South Carolina 29108 704-940-1228